



MULTI-POINT SEAT BELT

CROSS REFERENCE TO RELATED APPLICATIONS

This is a divisional application of the CA-serial number CA 2,313,780 related to an
5 international application number PCT/DE98/03270 (WO 99/24294, European Patent EP 1
037 773 B1, German Patent DE 197 49 780 C2) filed Nov. 10, 1998.

BACKGROUND OF THE INVENTION

1. Field of the Invention:

10 It is an object of the present invention to ensure the restraint of passengers of a transport
system, while preserving the user-friendliness, and to lower all acceleration-dependent forces
imposed on them in order to enhance the survival chance in the event of any accident (front-,
side-, rear-end collision and/or rollover or pile up/mass collision) or during in-flight
turbulence.

15 2. Discussion of the Prior Art:

It is known in the prior art to provide for a passenger of a transport system

- a three-point seat belt (safety belt or lap-shoulder seat belt assembly), mounted in the
motor vehicle, consisting of a shoulder belt extending across the upper body-part of his
body and of a lap belt extending across the lower body-part of his body; or
- 20 – a two-point seat belt, mounted in the aeroplane, acting as a lap belt extending across the
lower body-part of his body; or
- a suspender- (waist-) belt consisting of several pieces (belt-members).

In order to formulate in single terminology a generalized definition is presented for the
25 proper term:

Definition:

"Transport system"

"Stiff first transport-system
member"

Proper Term:

Motor vehicle or train or ship or aeroplane

Floor 6 of the transport system adjacent to a first seat-side
SR (Fig. 1) or seat-cushion frame at the first seat-side or
mid-tunnel (not drawn) of the motor vehicle adjacent to the

	first seat-side.
"Stiff second transport-system member"	Floor 6 of the transport system adjacent to a second seat-side SL or seat-cushion frame at the second seat-side or post section 91 (Figs. 13, 14) of the motor vehicle adjacent to the second seat-side or side rail of the motor vehicle adjacent to the second seat-side
"Stiff third transport-system member"	Floor 6 of the transport system adjacent to the second seat-side or seat-backrest frame at the second seat-side or post section adjacent to the second seat-side.
"Shoulder-belt deflector"	Belt deflector 5, 5b or D-ring 12 (Figs. 1, 13)

It is well known to provide different restraint systems in vehicles, predominantly, three-point seat belts in various types for seats. Evidently, when both shoulders of a passenger, conventionally belted, are not restrained in the event of an arbitrary collision with another

5 vehicle in any direction, shown in Figs. 3, 4 and 7, the unrestrained shoulder can always move and/or rotate freely, thereby resulting in severe/fatal injuries in real-world accidents when

- the head crashes into the steering wheel and/or window pane and/or
- the airbag crushes the head, which, loaded by the forces related to pitch-acceleration \ddot{U}_H , yaw-acceleration \ddot{O} , longitudinal and/or lateral acceleration, is in "oop" (out of position).

10 Moreover, by the definition of „submarining“ the belted passenger submarines (slips downward) under his seat belt thus negating the protective effect of the seat belt.

It is well known to provide two-point or lap seat belts for aeroplane seats as well as mid-portion of the rear seats of motor vehicles. This lap seat belt is far less effective than a three-point seat belt. Due to very large accelerations during a turbulence-related flight the protective

15 effect is very low.

A substantially improved protection is proposed by two different configurations of a one-piece seat belt, exemplified by DE 26 02 875 A1 (Figs. 8 to 10). An „X-shaped“ restraint is arranged by extending both shoulder belts crosswise over the upper body-part of body while the lower body-part of body is restrained by the lap belt. Each end of the one-piece seat belt is

20 connected to a belt retractor, fastened in the seat backrest. Two grab rings, positioned to the headrest, move along the belt. A single or double „X-shaped“ configuration is defined by

pulling a pair of grab rings and belt portions over the head, shoulders and head rest and engaging them in the corresponding hooks. Due to such intricate operation the seat belt remains unused.

According to US 3,977,696, US 5,123,673, US 5,411,319, DE-OS 23 45 847, DE-OS 28 13 888 and DE 196 29 878 A1 the restraint system comprises a three-point seat belt, a second shoulder belt and two belt retractors, responsible for retracting both belts. The „X-shaped” configuration, formed by extending both belts crosswise over the upper body-part of the body, has the following drawbacks in the event of an accident:

D1. Both belts are retracted to different length by two independently operating belt retractors within milliseconds.

D2. Under the load of the same belt force in a front collision the deformation of seat backrest, wherein both belt ends are fastened, is larger, thus increasing the forward motion.

Furthermore, it is impossible to attach an energy absorber because all four belt ends are occupied.

D3. Exemplified in US 5,123,673, the belt user has to depress two release buttons to release the respective main latch plates 9 from the main buckle assemblies. This two-click operation causes discomfort and hinders rescue work. See countermeasures by means of a single master release button, mentioned below.

A one-piece seat belt 1 (Fig. 1) ref. to DE-OS 28 13 888 is equipped with two belt retractors (not drawn), fastened to both belt ends in the seat backrest, and a belt deflector 17, anchored to the seat-cushion frame 3.3 of the mid-portion of rear seat. The feature, proposed for a child, has the following drawbacks:

D4. When the release button 84 is depressed, the first shoulder belt portion 1.1 gets entangled around the neck of passenger. For the operation of restraining and extending both belt portions into the „X-shaped” configuration, the passenger must lower his head first.

D5. Because all belt ends are occupied, it is impossible to attach energy absorbers and to adjust the belt to the size of an upper body-part of body 95 of an adult.

Generally, a child-seat is fastened by four auxiliary belts to the seat. Despite the „X-shaped” configuration of a one-piece seat belt to restrain a child, sitting in a child-seat, ref. to FR 2 342 872 A1 the problems, associated with the retraction of four auxiliary belts, submarining and energy absorption, remain unsolved in an accident.

Till now, trains, school buses and buses are not provided with restraint systems.

SUMMARY OF THE INVENTION

Accordingly, the principle object of the present invention is to provide for passengers of a
5 transport system seat belts, each equipped with a belt retractor, solely responsible for
retraction, blocking and tightening or for protraction, a lower belt deflector to loosely guide a
belt portion and multi-attachment points (multi-points of restraint), and to restrain every
passenger in multi-attachment points, in order to lower and distribute the acceleration-
dependent loads, shown in **Fig. 3** and **Tables 1 to 3**, to the multi-attachment points in the
10 event of any accident or during in-flight turbulence. Nowadays, belt tighteners are
incorporated into belt retractors, for example, of MB 500 SL in order to save costs, assembly
time and space.

A second object of the present invention resides in a user-friendly belt-feeding device to ease
the restraint and in a single master release button, which, when depressed, releases all latch
15 plates from the buckle assemblies and/or returns the belt-feeding device to the home (resting)
position. In emergency cases paramedics and fire-fighters can easily rescue the injured
passengers.

A third object of the present invention resides in the conventional three-point seat belt
associated with new parts, shown in **Fig. 2**, to serve as a transition product until multi-point
20 seat belts are put into production.

INDUSTRIAL APPLICABILITY

It should be apparent that the invention provides substantially improved restraint, including
25 the following features:

- a) The survival chance is enhanced by the restraint of
 - * both shoulders and the torso, when the passenger is thrown forward (**Fig. 4**, **Table 3**)
and/or subjected to the yaw \ddot{O} -acceleration-dependent torque T_{θ} , and
 - * both thighs and the lower body-part of the body to prevent submarining (**Fig. 12b**).
- 30 b) Because the belt retractor is attached to one belt end, a number of sets of vibration-
dampening energy absorbers ref. to US serial no. 09/554,464 (WO 99/24292,
PCT/DE98/03271, European Patent EP 1 037 771 B1, German Patent DE 197 58 498 C2,

CA pending patent 2,314,345) or German Patent DE 197 58 497 C2 can be attached to the other belt end (Figs. 11a, 11b, 15), thus gradually absorbing large impact energy below the respective injury-related values. The inventor of the present application has submitted those patent documents and applications to CIPO as well as USPTO. The vibration-dampening energy absorber consists of a number of clamping elements, having sites of predetermined fracture, and a retaining element, which, fastened to the seat-backrest frame and/or seat-cushion frame, can serve as an integral part thereof.

c) Owing to the different positions of pairs of upper buckle assemblies, in plug-in connection with the respective belt-detachable latch plates 25 (Fig. 16), passengers of different body proportions can adjust the belts by themselves. Moreover, the seats, equipped therewith, can be modified to be used by adults or children, thus increasing the rate of seat occupancy in a bus, train or an aeroplane, exemplified in Fig. 20.

d) In resting position the shoulder latch plate 2, in plug-in connection with an assisting buckle assembly 16, 16a, 16b, fastened to the seat cushion 3.1, B-, C-post section or seat backrest (Figs. 1, 2), is easily accessed by the passenger wanting to use the belt.

e) The seat belt can be equipped with a belt-feeding device, manually operated or by a drive apparatus, for example, hydraulic-piston cylinder unit, electrical motor (not drawn), which enhances the convenience and comfort of the user. This drive apparatus is switched on by a pressure sensor, built to the seat, or an existing switch such as lighting-, door- or touching switch. If the belt is not engaged within a dwell time, a control device is activated to switch off the drive apparatus and to reposition the belt-feeding device in the resting position.

f) For the convenience of the passenger, when stepping out, or for the quick-rescue of the injured passenger in accidents, the master release button 84 of the buckle assembly 9.1 is depressed to release all latch plates from the buckle assemblies and/or to return the belt-feeding device to the resting (home) position.

BRIEF DESCRIPTION OF THE DRAWINGS

A number of embodiments, other advantages and features of the present invention will be described in the accompanying tables and drawings with reference to the xyz global coordinate system:

Table 1 shows test data such as left / right thigh-force, belt force and pitch-angle of driver and co-driver in 50% offset crash test of several European vehicles at crash speed of 55 km/h.

Table 2 shows yaw angle O of driver / co-driver in a 50% offset crash tests.

Table 3 shows test data of the safest child-restraint system Chico Shuttle® at the converted velocity of 55 km/h in comparison with the safest vehicle among them listed in **Table 1**.

Fig. 1 is a perspective view of a seat with buckle assemblies attached to the seat backrest and seat cushion as well as of a 1st embodiment of a restraint system consisting of a multi-point seat belt **1**, shoulder-belt deflector **5**, D-ring **12**, latch plate **11** moveable along the lap belt, shoulder latch plate **2** of belt end portion, in the direction of arrow „Z” in plug-in connection with an upper buckle assembly **4**, and a seat belt in X-shape, formed by crossing the first and second shoulder belt portions **1.1**, **1.2**.

Fig. 2 is a perspective view of a seat and of a 2nd embodiment of a restraint system, comprising three-point seat belt **1e**, having a transition latch plate **2**, which will be inserted into a transition buckle assembly **4e** of a shoulder belt **1.11**, pulled in the direction of arrow „Z”.

Fig. 3 illustrates load cases I, II and III in z-y plane in the event of a real-world accident.

Fig. 4 is a perspective view of a restrained dummy thrown forward in VW Polo® in a 50% offset crash test.

Fig. 5 illustrates a yaw-acceleration \ddot{O} and yaw-angle O of a vehicle about the vertical axis „z_A” in a 50% offset crash test of two identical vehicles.

Fig. 6 illustrates a yaw angle O of vehicle about the vertical axis „z_A” in a 50% offset crash test into a stiff barrier.

Fig. 7 illustrates four collision types „U1” to „U4” ref. to the research work of Institute of Vehicle Safety, a Dept. of German Insurers Association.

Fig. 8 is a front view of a seat belt ref. to DE-OS 26 02 875 in the home position.

Fig. 9 is a front view of a double X-shaped seat belt ref. to DE-OS 26 02 875.

Fig. 10 is a front view of a single X-shaped seat belt ref. to DE-OS 26 02 875.

Fig. 11a is a schematic, perspective view of a 1st embodiment of a buckle assembly **4a**, equipped with release cable **4.2**.

Fig. 11b is a schematic, perspective view of a 2nd embodiment of a buckle assembly **4b**, equipped with an electrical release-motor **4.2b**.

Fig. 12a is a perspective view of a 1st embodiment of a belt-catching member **20.7a**.

Fig. 12b is a perspective view of a 2nd embodiment of a belt-catching member **20.7**.

Fig. 13 is a perspective view of a 1st and 2nd embodiment of a belt-feeding device and spatially-adjusting belt-feeding device **20a** from the resting position to the operative position and of a height-adjustable shoulder-belt deflector **5b**.

Fig. 14 is a schematic view of the 2nd and a 3rd embodiment of spatially-adjusting belt-feeding devices **20a** and **20b** in kinematics from the operative position to the resting position in x-y plane.

Fig. 15 is a schematic, perspective view of a seat backrest, equipped with a second belt retractor **13a**.

Fig. 16 is a schematic, perspective view of a belt-detachable U-shaped latch plate **25** and a 1st and 2nd embodiment of a height- and width-adjusting mechanism **27**, **27a**.

Fig. 17 is a cross-sectional view of the 1st embodiment of the height- and width-adjusting mechanism **27** along the line I-I of **Fig. 16**.

Fig. 18 is a cross-sectional view of the height- and width-adjusting mechanism **27** along the line II-II of **Fig. 17**.

Fig. 19 cross-sectional view of the 2nd embodiment of the height- and width-adjusting mechanism **27a** along the line I-I of **Fig. 16**.

Fig. 20 is a front view of the seat **3a to 3d**, in which the restraint systems **1a to 1d** are integrated, for passengers of different weights and body proportions (sizes).

DESCRIPTION OF THE PREFERRED EMBODIMENTS OF THE INVENTION

The advantages of the preferred embodiments in the Chap. "INDUSTRIAL APPLICABILITY" are outlined hereinafter with regard to the functions and features thereof.

The method of the present invention capitalizes on the premise that a seat belt is employed to restrain a passenger in at least four attachment points of the seat to distribute all acceleration dependant loads, particularly the yaw \ddot{O} -acceleration-dependent torque $T_{\ddot{O}}$, thereto in an accident, thus ensuring the operation of a single belt retractor to pre-tension (bias) as well as tension the belt, restraining both shoulders, upper- and lower body-part of the body and lowering all the loads, in particular, in co-operation with the energy-absorption when a number of sets of vibration-dampening energy absorbers is put into use. This will be apparent when all forces, imposed on the belted passenger, shown in **Figs. 3** and **4**, are

formulated in the event of a front collision, where the loads of the mass D_S of the torso are lowered because

- the forward motion „ w_v ” is minimized, thus substantially reducing the pitch-acceleration \ddot{U}_H and force F_{Hy} of the mass D_H of the head, and
- 5 – the yaw-acceleration \ddot{O} is minimized, thus substantially reducing the torque T_δ , imposed on the head. Great torque T_δ is the most latent force, responsible for sudden death.

To a great extent massive head injuries can be avoided.

10 Load case I in z-y plane: The rotating mass D_S rotates about the rotating axis „S” at the pitch-angle U_S and mass D_H about the rotating axis „z” at the pitch-angle U_H in **Table 1**, thereby resulting in the pitch-accelerations \ddot{U}_S , \ddot{U}_H and rotating forces F_{Sy} , F_{Hy} . The addition of both rotating forces yields the force F_V linked to the forward motion w_v of passenger, shown in **Fig. 4**.

15 Load case II in x-y plane: The upper body-part of body is subjected to the torque T_δ , exerted by the yaw-acceleration \ddot{O} about the rotating axis „z”. When the upper body-part is restrained in an X-shape, the torque is substituted by a pair of forces.

20 Load case III in x-z plane: The rotating mass D_S rotates about the rotating axis „S” at the rotating angle U_y and mass D_H about the rotating axis „z” at the rotating angle U_{Hy} , thereby resulting in the rotating accelerations \ddot{U}_y , \ddot{U}_{Hy} and rotating forces D_{Sy} , D_{Hy} (not drawn). In a rollover-accident the passenger is subjected to the load F_{Sz} .

25 Load case IV: In turbulence-related vibrations of an aeroplane the load D_{Sy} together with D_{Hy} takes the form of periodical load $\pm F_{Hx}$, F_{Sz} of $\pm F_{Sz}$, T_δ of $\pm T_\delta$, S_y of $\pm S_y$ and F_{Sy} together with F_{Hy} of $\pm F_V$.

30 The restraint system, illustrated in **Fig. 1**, is provided with a conventional belt retractor **13** having a clamping device, housed in the B-, C-, D-post section or in the seat backrest **3.2** at one of both seat-sides SL and SR of a seat and connected to the second belt end EL. The first belt end ER is provided with a shoulder latch plate **2**, which is retained, loosely guided by a lower belt deflector **17**, fastened to the vehicle floor, and inserted into one of the upper buckle assemblies **4**, **4a** to **4c**, **14**, **14a**, **18**, **18a**, **18b**, arranged in or to the seat backrest **3.2**. In all embodiments a main latch plate **9** can move along the seat belt **1** between both belt ends EL

and ER. When plug-in connecting the shoulder latch plate **2** (in the direction of arrow "Z") to the buckle assembly **4** and the main latch plate **9** to the main buckle assembly **9.1**, an X-shaped restraint of the upper body-part of body and both shoulders as well as a restraint of the lower body-part of body are accomplished by the first and second shoulder belt portion **1.1**, **1.2** and the lap belt portion **1.3**.

In the 2nd embodiment, shown in **Fig. 2**, a transition product, comprising a conventional three-point seat belt **1e** and new parts, has to be invented due to the delay in producing multi-point seat belts **1**. The floor fitting (not shown) is replaced by the lower belt deflector **17**. The first belt end of the lower first shoulder belt portion **1.11** is provided with transition latch plate **2**. The first belt end of an upper first shoulder belt **1.12** and the second belt end are equipped with a transition buckle assembly **4e**, having a transition release button **84c**, and with a second belt retractor **13a**, arranged in the seat backrest **3.2**. Due to the second belt retractor the transition buckle assembly **4e**, acting as the shoulder latch plate **2**, **2a** of multi-point seat belt, is located in a home position on a seat-backrest aperture of the seat at the first seat-side.

Hence, the seat-design is not compromised. In a coupling position the restraint in an X-shape is defined by plug-in connection of transition latch plate **2** with the transition buckle assembly **4e**, pulled out from the seat-backrest aperture, wherethrough a transition portion of the upper first shoulder belt is projected. This transition portion and the lower first shoulder belt portion **1.11** define the first shoulder belt portion **1.1**. In order to resolve the above-mentioned drawback **D1**, the spring force of the second belt retractor **13a**, to retract the upper first shoulder belt **1.12**, released by depressing the transition release button **84c**, is far less than that of the belt retractor **13**. Although the circumference of the restrained passenger varies, depending on the clothes worn, and the seating-position differs the lower first shoulder belt portion **1.11** always projects through the lower belt deflector **17** at a sufficient length of " l_1 " to maintain the function of the belt retractor **13** to retract, to block the belt as well as to release the retracted belt during the journey and the function of the belt tightener (not drawn), incorporated in the belt retractor, to forcefully retract (withdraw) and tighten the belt in an accident. The transition release button **84c** of transition buckle assembly **4e**, arranged to or in the seat, can be controlled neither by release cable **4.2** nor by electrical release-motor **4.2b**. It can only be activated by electrical signals emitted from the master release button **84** when depressed.

The second belt end of upper first shoulder belt **1.12** can be connected either to a coupling fitting **1.2a, 1.2b** (Figs. 11a, 11b, 15) or to the second belt retractor **13a** having a coupling fitting **1.2b** (Fig. 15) in order to receive a number of vibration-dampening energy absorbers to dissipate great impact energy and dampen strong vibration.

- 5 In another embodiment an upper first shoulder belt **1.12a** consists of the transition buckle assembly **4e** and a shoulder latch plate **2a** (not shown), similar to latch plate **2** (Fig. 1), which is plug-in connected to
- the upper buckle assembly **4, 4a to 4c, 14, 14a, 18, 18a, 18b, 18.1 to 18.3**, arranged to the seat backrest, in operative position or
 - 10 – the assisting buckle assembly **16, 16a, 16b** in resting position.

When motor vehicles are already licensed, modification of different seats and three-point seat belts can easily be accomplished by arrangement of at least one buckle assembly, the lower belt deflector **17**, the second belt retractor **13a** and by a variety of one-piece, detachable, upper first shoulder belts **1.12a** with different lengths. Furthermore, the latch plate **2a** can be

15 detached from the buckle assembly by depressing the master release button **84**.

A first shoulder belt portion **1.1** is defined by the upper first shoulder belt **1.12a** and the lower first shoulder belt portion **1.11**.

With an expensive modification or in new transport system the convenience and comfort are enhanced by the use of belt-feeding device **20, 20a to 20d**, where the upper first shoulder belt

20 **1.12, 1.12a** with transition buckle assembly **4e** is a part of the belt-feeding device.

Beyond doubt, the three-point seat belt **1e** in plug-in connection with the upper first shoulder belt **1.12, 1.12a** is suited as a temporary solution for the multi-point seat belt **1, 1a to 1d**.

In the above-mentioned embodiments to resolve the above-mentioned drawback **D4** the upper body-part of body is restrained by extending the shoulder belt portions crosswise in an

25 X-shape

- c1) when at least one shoulder latch plate **2** is plug-in connected to the upper buckle assembly of the seat backrest; or
- c2) when a shoulder latch plate **2**, arranged to the first belt end ER of the first shoulder belt portion **1.1** of a belt-feeding device **20a, 20b**, is plug-in connected to the upper buckle
- 30 assembly of the seat backrest.

The feature ref. to c2) has the advantage that the common practise of operating the conventional three-point seat belt is preserved.

In order to resolve the above-mentioned drawbacks **D2** and **D5** great energy is absorbed and strong vibration is dampened by a large number of vibration-dampening energy absorbers connected to the respective upper buckle assemblies **4, 4a to 4c, 4e, 7, 8, 8a to 8d, 9.1, 14, 14a, 15, 15a, 18, 18a, 18b, 18.1 to 18.3, 19, 19a, 19b, 19.1 to 19.3** (Figs. 1, 20) to which
5 latch plates are plug-in connected.

The lower belt deflector **17** comprises a housing having an attachment hole to receive a pin **17.1**. Both members can be made in one piece. If necessary, the pin **17.1** is surrounded by a sleeve **17.2** of plastics, having corrugation or knobs, which is a common part of the conventional D-ring **12**. This D-ring **12** can be replaced by the lower belt deflector **17**. The
10 aperture of the belt deflector **17** to loosely guide the belt portion is dimensioned so as to retain the latch plate **2** in resting position, thus allowing the use as a three-point seat belt.

To prevent the entanglement of the first shoulder belt portion **1.1** behind the seat, particularly when positioned furthest forward, that first shoulder belt portion **1.1** in resting position is intercepted by the belt-catching member **20.7, 20.7a** (Figs. 12a, 12b). When the
15 second shoulder belt portion **1.2** and the extending belt portion **1.4** are arranged to the post section, both shoulder belt portions can also be intercepted by the belt-catching member.

When the seat **3c** (Fig. 20) has a high seat backrest, the curved guide tube **20.1** of belt-feeding devices **20a** (Fig. 13) can be modified to a straight-running operating arm **20.2** of the belt-feeding device **20**.

20 In the 2nd or 3rd embodiment the belt-feeding device **20a** or **20b** is provided with a height-adjustable belt housing **20.4a** and radial-adjustable tube **20.3** (Figs. 13, 14). Both devices differ from each other by the position of the guide tubes **20.1** on the seat backrest. Each guide tube can be driven by a drive apparatus, housed in the seat backrest. The guide tube **20.1** of the belt-feeding device **20a** is pivotally attached in a stiff supporting tube **3.61** of the height-
25 adjustable head rest **3.6a**.

The height of „ Δh ” of belt housing **20.4a**, having a latch plate **2**, plug-in connected to any buckle assembly **4, 14, 18**, is adjustable when the passenger moves two openings, facing each other, along the operating arm **20.2a**. Alternatively, the passenger can move a handle **5.2**, such as locking handle **27.5** of the height- and width-adjusting mechanism **27, 27a** (Figs. 13, 17 to
30 19), to adjust the height of „ Δh ” of the shoulder-belt deflector **5b**.

In order to ensure the operation of pro- and retracting any shoulder-belt portion, arranged in the seat backrest (Figs. 8 to 10), is loosely guided by a shoulder-belt deflector which, having a rectangular shape, is usually pressed in a seat-backrest aperture of the seat backrest on the top edge.

5 The belt-feeding devices **20a**, **20b** have to meet the following criteria:

- Passengers can freely get in and out of the vehicle compartment thanks to the distances of „a” and „b” between the post section **91** and operating arm **20.2a** (Fig. 14) in resting position; and
- the device, when moved, does not interfere with the head rest **3.6a**, height-adjustable about „ Δh_K ”, and with the head of the passenger with/without hat **92**.

10 Regarding the kinematics of the height-adjustable belt housing **20.4a** with the latch plate **2** from the operative position to the resting position, the trajectories of „Ba2” and „Bb” are well clear of the passenger's head thanks to a radial-adjustable tube **20.3** incorporated into the operating arm **20.2a**. Without the radial-adjustable tube **20.3** the operating arm in the trajectory of „Ba1” would interfere with that hat.

15 Upon plug-in connection of the latch plate **2** with the buckle assembly **4**, **4a**, **4b** the belt end ER of belt portion **1.1** is connected to the coupling fitting **1.2a**, **1.2b** (Figs. 11a, 11b), whereto a number of energy absorbers is attached to absorb energy. In a cost-saving embodiment without the latch plate **2** and buckle assembly, the belt end ER of belt portion **1.1** is directly connected to the coupling fitting **1.2a** or **1.2b** (Fig. 15) to receive energy absorbers, the retaining elements of which are fastened to the seat backrest frame **3.4d**. In order to absorb great energy and damp strong vibration during in-flight turbulence or in the accident of a fast speeding car or high-speed train, the belt retractor **13**, coupling fitting **1.2b** of which is connected to energy absorbers, is moveably attached to the oblong holes of a stiff plate **13.3**, fastened to the seat-backrest frame at the seat-side SR so that the other belt end EL can be exploited to receive additional energy absorbers. In excess of threshold value the belt retractor pulls the clamping elements along the respective retaining elements to absorb energy and damp vibration.

20 In the 1st to 3rd embodiment (Figs. 11a, 11b, 18) the buckle assembly **4a**, **4b**, **4c** is form- and/or force-locking connected to the seat-frame of the seat.

For the convenience of the passenger when egressing from the vehicle and in cases of emergency the following embodiments of detachment are proposed:

To disconnect the latch plates 2, 11 and/or 25 from the buckle assemblies 4, 14, 14a, 15, 15a (Fig. 1) and pairs of supplement upper buckle assemblies 18 / 19, 18a / 19a, 18b / 19b, 18.1 / 19.1 to 18.3 / 19.3 (Fig. 20) of the seat arrangement, particularly for children, as well as from the anti-submarining buckle assemblies 7, 8, 8a to 8d (Figs. 1, 12b), the master release button 84, when depressed, activates the release cables 4.2 and/or electrical release-motors 4.2b, which pull the release button 84a and/or 84b of the buckle assemblies (Figs. 11a, 11b, 18). When depressing the master release button 84 the drive apparatus of the belt-feeding device 20, 20a, 20b returns the first shoulder belt portion 1.1 from the operative position to the resting position.

In the 1st embodiment (Figs. 17 to 19) the height- and width-adjusting mechanism 27 comprises a frame 29, buckle-assembly unit 18.3, 19.3, a pair of tubes 27.4, members 27.5 to 27.9 and a pair of tubes 27.1 having a plurality of vertical locking slots, in form- and force-locking connection with an angle fitting 26a. The frame 29 consists of a pair of outer tubes 27.3, a pair of tubes 27.2 and a connecting member of all tubes. The locking handle 27.5 is form- and force-locking connected to the slots of the inner tubes 27.4.

These inner tubes 27.4, inserted into the outer tubes 27.3, are pre-loaded by the tube-springs 27.6. Each tube-spring 27.6 on a sleeve 27.7, secured by pin 27.8, protruding through the holes of the inner tube 27.4, presses against the spring rest 27.9 of the outer tube 27.3.

The locking handle 27.5 is in engagement with a pair of vertical-locking slots of tubes 27.1. The locking handle 27.5, when pulled out from both slots, is detached therefrom. The height of mechanism 27 and buckle assembly can be adjusted

The outer tube 27.3 is provided with a plurality of horizontal locking slots q, r, s etc., drawn with dotted lines, shown in Figs. 17, 19.

After the pawl 18.10, pre-loaded by the pawl-spring 18.5, is detached from the horizontal locking slot r by its movement in the direction of arrow (Fig. 18), the housing 18.12 of the buckle-assembly unit 18.3, 19.3, form-locking connected to the upper buckle assembly 4c thereof, can be moved along both outer tubes 27.3.

Belt-detachable U-shaped latch plates 25 offer the passengers a feature to adapt their body proportions to the appropriate pair of supplement upper buckle assemblies into which the latch plates 25 are inserted (Figs. 16, 20). Any belt portion, such as 1.1, 1.2, is loosely guided

thereby, secured by a quick-release pin 25.1 thereof and detached therefrom by pulling the quick-release pin. To adapt a small body proportion of, say, a child, far lower than the upper buckle assembly 4 suited for adults, at least one pair of belt-detachable latch plates 25 are plug-in connected to one of the pairs of supplemental upper buckle assemblies 18 / 19, 18a / 19a, 18b / 19b, 18.1 / 19.1 to 18.3 / 19.3, arranged to the seat backrest at the first and second seat-side (Figs. 1 and 20). For safety reasons and easy access the belt-detachable latch plates 25, when not being used, are stored and secured in a storage box 25.5 of the seat (Fig. 20).

For juxtaposed seats in vehicles, buses, trains and aeroplanes it is recommended to use a single locking handle 27.5 to operate the 2nd embodiment of the height- and width-adjusting mechanism 27a of each seat 3c, having, for example, three pairs of openings 18.1 / 19.1 to 18.3 / 19.3 to receive a pair of shoulder latch plates (Figs. 19, 20).

The frame 29a consists of two pairs of outer tubes 27.3, two pairs of tubes 27.2, a pair of connecting members of all tubes and members 18.3, 19.3, 27.6 to 27.9a, 27.11, attached to the outer tubes 27.3.

The locking handle 27.5 is form- and force-locking connected to slots of the inner tubes 27.4 by the pins 27.12. After inserting these inner tubes into the outer tubes 27.3 the locking plate 27.10 is form- and force-locking connected to the slots of the inner tubes and to the pins 27.12.

After securing the spring rest 27.9a by the retaining rings 27.11 and both sleeves 27.7a by the pins 27.8, protruding through the holes of inner tubes 27.4 and oblong holes of outer tubes 27.3, the inner tubes with locking handle 27.5 are pre-loaded by tube-springs 27.6. The locking handle 27.5, when pulled out from both slots, is detached therefrom. The height of height- and width-adjusting mechanism 27a can be adjusted.

Although the present invention has been described and illustrated in detail, it is clearly understood that the terminology used is intended to describe rather than limit. Many more objects, embodiments, features and variations of the present invention are possible in light of the above-mentioned teachings. Therefore, within the spirit and scope of the appended claims, the present invention may be practised otherwise than as specifically described and illustrated.